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Specification Amendments

The paragraph beginning on line 11 of page 4 has been amended as follows:

Figure 3A is a longitudinal cross-sectional view of an alternative embodiment of the gate valve of [Figure 2] the present invention shown in the closed position;

The paragraph beginning on line 13 of page 4 has been amended as follows:

Figure 3B is a longitudinal cross-sectional view of the gate valve of Figure [2] 3A shown in the open position;

The paragraph beginning on line 15 of page 4 has been amended as follows:

Figure 3C are radial cross-sectional views of three alternative gate components of the gate valve shown in [Figure 2] Figures 3A and 3B; and

The paragraph beginning on line 1 of page 5 has been amended as follows:

Referring to Figure 1, a preferred embodiment of the gate valve 10 is shown installed in an exemplary flow completion system 12. As explained more fully in applicants' co-pending U.S. Patent Application No. [[FMC Docket No. 66-12272]] 09/815,437, which is hereby incorporated herein by reference, the flow completion system 12 comprises a wellhead housing 14 which is installed near the top of a well bore (not shown), a tubing spool 16 which is connected over the wellhead housing 14 and which includes a central bore 18 that extends axially

therethrough, and a tubing hanger 20 which is supported in the central bore 18 and which includes a generally annular body 22 having an outer, stepped cylindrical wall 24. The tubing hanger 20 supports at least one tubing string 26 which extends into the well bore and defines a tubing annulus 28 surrounding the tubing string. In addition, the tubing hanger 20 includes a concentric production bore 30 which communicates with the tubing string 26, a lateral production passageway 32 which extends between the production bore 30 and a production outlet 34 in the tubing spool 16, an annulus bore 36 which extends generally axially between the tubing annulus 28 and a portion of the central bore 18 located above the tubing hanger, and a number of service and control conduits 38 which extend generally axially through the tubing hanger. The flow completion system may also comprise one or more valves 40 for controlling flow through the production outlet 34, and a controls bridge 42 for connecting the annulus bore 36 and the service and control conduits 38 with corresponding external service and control lines (not shown).

The paragraph beginning on line 16 of page 6 has been amended as follows:

Referring to Figure 2, substantially all of the operational components of the gate valve are housed entirely within the body 22 of the tubing hanger 20. In addition, the gate valve 10 is oriented generally axially within the body so as to occupy a minimum of the radial cross sectional area of the tubing hanger. In order to most readily accommodate this vertical orientation of the gate valve 10, the annulus bore 36 preferably includes a lateral branch which is connected to a

longitudinal branch, and the gate valve is disposed within a gate cavity that is positioned between the branches. For example, in the embodiment of the invention depicted in Figure 2, the annulus bore 36 is shown to comprise a [first] lower lateral branch 54 which extends generally laterally through the body 22, a [second] lower axial branch 56 which extends generally axially through the body between the [first] lower lateral branch and the tubing annulus, [a third] an upper lateral branch 58 which extends generally laterally through the body, and [a fourth] an upper axial branch 60 which extends generally axially through the body between the [third] upper lateral branch and the portion of the central bore 18 that is located above the tubing hanger.

The paragraph beginning on line 8 of page 7 has been amended as follows:

The gate valve 10 will now be described with reference to Figures 3A through 3C, which depict a slightly modified version of the gate valve from the embodiment shown in Figure 2. Referring to Figure 3A, the gate valve 10 is shown to comprise a gate cavity 62 which is formed in the body 22 of the tubing hanger 20 and is connected to the [first and third] branches 54[,] and 58, two preferably floating seats 64, each of which is mounted in a respective seat pocket 66 that is formed in the body at the intersection of each branch 54, 58 with the gate cavity, a gate 68 which is slidably disposed within the gate cavity adjacent the seats, and an actuating mechanism 70 which is positioned in a service and control conduit 38 that is oriented generally vertically over the gate cavity. The gate 68 includes a generally flat face 72 which is adapted to sealingly engage the

seats 64 when the gate valve 10 is in the closed position. In addition, the gate 68 may have any of a variety of transverse cross-sectional configurations, including any of the configurations depicted in Figure 3C.

The paragraph beginning on line 20 of page 7 has been amended as follows:

The gate 68 also comprises a generally lateral first flow port 74 which extends from the face 72, a generally longitudinal second flow port 76 which is connected to the first flow port, and, in the embodiment of the invention depicted in Figure 3A, a generally lateral third flow port 78 which extends between the second flow port and the face 72. Thus, the second flow port 76 is divergent from the first flow port 74, and the third flow port 78 is divergent from the second flow port 76. In the context of the present invention, this means that the axis of the second flow port 76 is offset angularly relative to the axis of the first flow port 74, and the axis of the third flow port 78 is offset angularly relative to the axis of the second flow port 76. In the embodiment of the invention shown in Figure 3A the amount of angular offset between both the second and first flow ports and the third and second flow ports is approximately ninety degrees. In this manner, the flow ports function to change the direction of fluid flow through the gate 68 by 180 degrees.

The paragraph beginning on line 10 of page 8 has been amended as follows:

The actuating mechanism 70 functions to move the gate 68 between a valve open position, in which the first flow port 74 is aligned with the [first] lower

lateral branch 54 and the third flow port 78 is aligned with the [third] upper lateral branch 58, and a valve closed position, in which the first and third flow ports 74, 78 are offset from the respective [first and third] branches 54[,] and 58. In the embodiment of the invention depicted in Figure 2, the actuating mechanism 70 is similar to that described in applicants' co-pending U.S. Patent Application No. [[66-12273]] 09/815,436, which is hereby incorporated herein by reference.

Thus, the actuating mechanism 70 comprises a piston head 80 which sealingly engages the service and control conduit 38, a valve stem 82 which is connected between the piston head and the gate 68, and a mechanical return biasing mechanism 84, such as a stack of Belleville washers, that is operatively engaged between the piston head 80 and the body 22 of the tubing hanger 20.

The paragraph beginning on line 1 of page 9 has been amended as follows:

In the embodiment of the invention shown in Figure 3A, the actuating mechanism 70 includes a piston 86 which sealingly engages the service and control conduit 38 and which is connected to the gate 68. The actuating mechanism 70 also comprises a mechanical return biasing mechanism, such as a spring 88, which is operatively engaged between the gate 68 and the body 22 to urge the gate into, for example, the closed position. The spring 88 may be positioned between the bottom of the gate 68 and the bottom of the gate cavity 62, in which event the actuating mechanism 70 preferably also includes a stop sleeve 90 to limit the downward movement of the gate 68. Of course, the actuating mechanism, including the return biasing mechanism, could be any

mechanical, electrical or hydraulic mechanism that is currently known by those skilled in the art to be useful for moving the gate between its open and closed positions.

The paragraph beginning on line 3 of page 10 has been amended as follows:

In the closed position of the gate valve 10, which is shown in Figure 3A, the gate 68 will seal off the annulus bore 36 by preventing fluid flow between the lower lateral [first] branch 54 and the upper lateral [second] branch [56] 58. In order to ensure that the gate 68 sealingly engages the seats 64, the gate valve 10 may include a load reaction disc 98 which is threadedly received in a corresponding orifice in the cover plate 94. Accordingly, the load reaction disc 98 may be tightened against the gate 68 to ensure that it engages the seats 64 with sufficient force to prevent fluid from leaking past the seat-to-gate interface.

The paragraph beginning on line 11 of page 10 has been amended as follows:

When it is desired to open the gate valve, pressurized hydraulic fluid is introduced into the service and control conduit 38 above the piston 86 through, for example, a male coupling 100. The hydraulic fluid will force the piston 86 downward, and the piston will in turn push the gate 68 downward until the gate contacts the stop sleeve 90. In this position, which is shown in Figure 3B, the lateral first and third flow ports 74[,] and 78 are aligned with the lateral [first and third] branches 54[, 56] and 58, respectively, and the annulus bore 36 is therefore opened. In order to return the gate valve to the closed position, the

hydraulic fluid is removed from above the piston 86, thus allowing the spring 88 to push the gate 68 upward until it contacts the top of the gate cavity 62.

The paragraph beginning on line 21 of page 10 has been amended as follows:

Referring now to Figure 4, another embodiment of a gate valve of the present invention, which is indicated generally at 110, is shown installed in a tubing hanger 20. In this embodiment, the annulus bore 36 of the tubing hanger 20 includes a [first] lateral branch 112 which extends generally laterally through the body 22 from the gate cavity 62, a [second] lower axial branch 114 which extends generally axially through the body between the [first] lateral branch and the tubing annulus, and [a third] an upper axial branch 116 which extends generally axially through the body between the gate cavity and the portion of the central bore 18 that is located above the tubing hanger. In addition, the gate valve 110 of this embodiment comprises a single seat 64 which is mounted in a seat pocket 66 that is formed at the intersection of the lateral [first] branch 112 with the gate cavity 62, and a gate 118 which is slidably disposed in the gate cavity 62 against the seat 64. The gate 118 includes a front face 120 which sealingly engages the seat 64, a generally lateral first flow port 122 which extends from the front face, and a generally longitudinal second flow port 124 which extends between the first flow port and the top of the gate. In this manner, the gate will change the direction of the fluid flow by 90 degrees.

The paragraph beginning on line 15 of page 11 has been amended as follows:

The gate valve 110 also comprises an actuating mechanism 126 for moving the gate 118 between an open position, in which the lateral first flow port 122 is aligned with the lateral [first] branch 112, and a closed position, in which the first flow port is offset from the [first] lateral branch. The actuating mechanism 126 includes a piston 86 which sealingly engages a bore 128 that is positioned below the gate 118, a conduit 130 for connecting the bore with a source of negative pressure, and a return biasing mechanism, such as a spring 88. Of course, the actuating mechanism 126, including the return biasing mechanism, can be any such mechanism disclosed herein or in applicants' above-mentioned co-pending U.S. Patent Application No. [[FMC Docket No. 66-12273]] 09/815,436.

The paragraph beginning on line 3 of page 12 has been amended as follows:

When the gate valve 110 is in the closed position, which is shown in Figure 4, the gate 118 will seal against the seat 64 and prevent fluid from flowing between the [first and third] branches 112[,] and 116 of the annulus bore 36. As in the previous embodiment, the gate valve 110 preferably includes a load reaction disc 98 for urging the gate 118 into tight sealing engagement with the seat 64. In order to open the gate valve 110, the bore is exposed to the source of negative pressure via the conduit 130. The negative pressure will consequently draw the piston 86, and thus the gate 118, downward into the open position. When it is desired to again close the gate valve 110, the negative

pressure is removed from the bore 128, thus allowing the spring 88 to push the gate upward into the closed position.